

## SPECIFICATION AMENDMENTS

The specification at paragraph [0033] has been rewritten as follows:

[0033] Referring specifically to Figs. 3A, 7 and 8, a valve seat member 34 is preferably made out of plastic and is press fit over the valve seat receiving area 32. By this arrangement, fluid passage is allowed axially between the buttresses 30, while side loads on the valve seat member 34 are directed to the thicker base of the ribs where they are better absorbed. An alignment shelf 36 is preferably provided on the fluid control valve body 12 for providing proper depth of alignment of the valve seat member 34. A ball valve 38 is preferably held between the valve seat 40 and the valve ~~retainer~~ seat receiving area 32. The valve seat member 34 preferably provides a passageway 42 to the fluid control passage 20. The ball valve 38 is preferably operable to selectively cut off supply of flow from the pressure supply channel 18 to the pressure control passage 20.

The specification at paragraph [0034] has been rewritten as follows:

[0034] The valve seat member 34 preferably includes three wing members ~~33~~ 35 which extend radially therefrom for securing the valve seat member 34 between the lower surface of the pole piece 62 and the shelf 36. In accordance with a preferred embodiment of the present invention, ~~it~~ it is generally necessary to provide a plastic part of very close tolerances such that the distance between the valve seat 40 and the upper surface of at least one, and more preferably all, of the wing ~~portion~~ members 35 ~~is~~ are maintained ~~in~~ to a tolerance of  $\pm 0.025$  mm or better.

The specification at paragraph [0036] has been rewritten as follows:

[0036] Referring specifically to Figs. 3, 6, 9 and 10, an armature 60 is preferably provided, which fits within the wall 54 and is axially movable in response to a current in the coil 46. The pole piece 62 is secured between the lower portion of the bobbin 48 and the fluid control body 12. The pole piece 62 preferably includes a center orifice 64, which allows the control rod 66 to actuate the ~~valve~~ ball member 38. In accordance with a preferred embodiment of the present invention, the armature 60 is preferably provided with at least one, and more preferably at least two, semi-circular pressure relief vents 68 formed in the lower portion of the armature 60 so as to preferably control pressure build up and prevent extra pressure during solenoid actuations. This beneficially increases net force and improves precise solenoid performance.

The specification at paragraph [0038] has been rewritten as follows:

Referring specifically to Fig. 14, there is shown an on/off type solenoid control valve 100, in accordance with an alternative embodiment of the present invention. Control valve 100 preferably includes a fluid control body 112 and a solenoid portion 114. Similar to the above embodiment, a plastic valve ball retaining portion is shown at 134 engaging a ~~feed tube~~ casing member 150, and holds the unit together to the fluid control body 112. However, in this embodiment, a separate flux tube member 156a is preferably provided, rather than the integral flux tube as shown in the previously described embodiment. The flux tube member 156a preferably includes a support structure 180, which supports a bearing 182 therein. The stepped bobbin arrangement

152 is also similar to the previously described embodiment. Differing in this embodiment, however, is an armature 160, which preferably includes an actuating rod 166, which preferably extends through the entire core of the armature 160 and exits along the axis A-A on the side opposing the valve member 166a. The bearing 182 of support structure 180 preferably slidably guides the rod 166 during actuation of the armature 160. This allows for back end guidance adjacent the bobbin 152. With this structure in place, the flux tube member 156a can be closer to the armature 160, which provides improved magnetic characteristics, particularly in high-pressure solenoid control valve applications.